

Possibilities of Heat Pump Integration for the Renovation of Dwelling Houses

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Abstract – The technical solutions for the installation of heat pumps in individual houses are well known, but its integration in the existing systems in dwelling houses is not common. Heat pump technology is referred to as renewable but would have technical, economic and environmental impact on the whole existing heat supply system in a dwelling house. The aim of this article is to investigate the possibility of using heat pumps for supplying heat to the existing residential buildings.

This article examines the possibilities to supplement the engineering systems with additional heat pumps. The smallest heat pump end-user group is the dwelling stairwell. The possibility to use heat pumps in a separate apartment has not been analysed.

This article analyses the integration of heat pumps for residential heat supply in the building. The primary heat source is the exhaust air or wastewater. All calculations have been made for several real existing dwelling houses in Birštonas town (Lithuania) within the framework of the CONCERTO Eco-Life project. The analysis also provides economic and environmental assessment of the alternatives.

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Keywords – Heat pump, dwelling house renovation, technical, economic and environmental assessment

I. INTRODUCTION

Energy efficiency of buildings is one of the leading topics of the European Union policy. Two different aspects are defined: renovation of the existing buildings and construction of new buildings. In both aspects the greatest attention is paid to the renewable energy resources and efficient transformers of fossil fuel energy. Both these aspects are unified by heat pumps (HP), though they are not apparently a widely used technology for heat supply to residential buildings. This article analyses the possibility of HP integration for heat supply in renovated dwelling houses in Lithuania. The specific feature of heat supply of these houses – the main part of heat is supplied from the district heating system (DHS). The possible alternatives of building service systems (heating, ventilation, domestic hot and cold water) are analysed in their energy audits and feasibility studies. This article examines the possibilities of HP integration in the renovated building services systems. The smallest group of HP end-users is a stairwell of a four-storey residential building. The possibilities to integrate HP in a separate apartment or buildings' disconnection from DHS is not the topic of this article.

HP systems offer economic alternatives for recovering heat from different sources for use in various industrial, commercial and residential applications. A review of the major hybrid HP systems suitable for application with various heat sources is presented [1]. The evolution of new hybrid systems has also enabled HP to perform efficiently with wider applications and markedly reducing carbon emission. The ventilation, heating and cooling of a building can be provided by advanced mechanical ventilation heat recovery systems which incorporate HP [2, 3]. Different systems with combinations of solar collectors and ground-source HP are simulated and optimized [4, 5]. A novel heating and cooling concept is developed to make use of rainwater for HP of HVAC systems. [6, 7]

In FW6 and FW7 Concerto projects or countries with cooler climate, solutions with HP are also applied.

One of the objectives of the project Concerto Class1 is “to optimize the integration of low-energy building technologies with supply and distribution technologies”. HP is planned to cover about 26% of the required 2.7 MWh of heat in Stenløse community. Energy demand for heating in one group of buildings reaches 30-34 kWh/m² and in the other one – about 16 kWh/m² [8].

One of the scientific research and innovation activities of the project Concerto Green Solar Cities is “combination of large-scale solar heat device with low temperature micro network, integrating HP into network and increasing the general effectiveness of solar device”. In Salzburg (Austria), the project of 2000 m² of solar collectors is planned with 200 m³ accumulation tank and integrated HP [9].

HP integration possibilities are widely discussed in the Intelligent Energy Europe project Promotion of Efficient Heat Pumps for Heating (ProHeatPump) [10]. While performing this work, the information of the Intelligent Energy Europe Sento project was methodically useful [11]. In the context of the analysed problem, the research results for the Nordic countries are noteworthy [12]. In these researches, the life cycle analysis is used. It is confirmed that the Ecodesign concept for energy consuming and related products [13] is widely used in many spheres voluntarily, and it is likely to become obligatory for buildings.

Solar thermal energy and HP combination in one system seems to serve the purpose. The results of the solar heat system's combination with other generators in nine buildings [14] are presented, in four of which HPs are integrated. Thorough research shows that exhaust air HP needs less additional investments than air-water HP or ground (vertical

bore) HP [15]. In Germany, 40-50 % of all passive houses use the integrated HP system for heating and domestic hot water [16]. Usually, heat source is exhaust ventilation air, often combined with preheating of the supplied air or cooling in a ground heat exchanger [17]. Nowadays in market very high efficiencies of heat recovery from the exhaust air are expected. Such efficiencies are reached in standard conditions and are far from the seasonal efficiency which the end-user is concerned about. It should be noted that in the mentioned estimations the seasonal efficiency is defined in three values (30, 55 and 70 %) and thus, it poses high confidence [15].

Besides air, another warm flow outgoing from a building is waste water (WW). A very well structured information on the possibilities of waste water heat usage is found in [18]. For example, in Switzerland, about 2 TWh of heating of premises could be covered by WW [19], and it makes 7 % of the country's heat demand for heating. It should be noted that, in the future, after the thermal characteristics of building will be gradually improved, the flow of air and warm water from the building shall change very slightly, and thus should receive more attention.

II. ESTIMATED SELECTION OF HP ALTERNATIVES

In this article, a slightly modified and amended SENTRO renewable energy evaluation tool [11] was used for the selection of HP alternatives. The tool provides the possibility of selection which is assessed according to technical, financial, organizational and environmental parameters. The weight coefficients given for these parameters help to compare the probability of the solution with the effort required to implement it. When the indicator is less than 33 %, the solution is evidently unacceptable, and, when it reaches 75 %, the solution is worth paying attention to. Mediate values might have various descriptions, but the selection of such alternatives is more or less arguable and needs a more thorough argumentation.

Two separate HP groups are estimated – ground source HP (G) and waste heat HP (R). In the first case, HP needs a heat collector in the ground. In the second case, HP uses waste heat of the maintained building, which normally is exhaust air (ExAir) or WW. The aggregated indicators of these two groups (G and R) according to SENTRO are presented in Table I.

TABLE I
AGGREGATED INDICATORS OF HEAT PUMP TECHNOLOGY ASSESSMENT

Group	HP technology	Technical parameters	Financial parameters	Organisational parameters	Environmental parameters	Success probability
HP_G	Ground HP for heating	61%	75%	58%	33%	57%
HP_R	Other (not ground) HP	89%	75%	75%	33%	67%

The technical parameters cover the assessment of ground surface suitability, exhaust ventilation, demand of heat and domestic hot water, type of heating system and HP area.

The financial parameters cover the assessment of investment and operational maintenance costs, the system's life cycle costs and an appropriate subsidy scheme.

The organizational parameters include the assessment of the drilling and/or construction permit, quality of the system, sufficient system supply and qualified staff.

The environmental parameters cover the assessment of impact to global warming. According to the size of buildings, for a successful HP, installation HP_R alternatives are further analysed.

III. POSSIBLE TECHNICAL ALTERNATIVES

The article examines the possible technical alternatives of HP installation in the services systems of renovated four-storey dwelling houses with 40 apartments built in 1970. They have a water to water plate heat exchanger, natural ventilation system, direct/instantaneous domestic hot water system (DHW) with recirculation. So, the reference alternative is when HPs are not installed. The technically possible alternatives, when the building's main heat demand is supplied by DHS, are presented below.

HP_R1. Usage of heat recovered from mechanical exhaust air for HP, when there is no mechanical air supply system

and HP is installed in a stairwell's bivalent (together with DHS) semi-accumulation domestic hot water preparation system.

HP_R2. Usage of heat recovered from mechanical exhaust air for HP, when there is a mechanical air supply system and HP is installed in a stairwell's air handling unit instead of air preheating section after or instead of plate recuperator.

HP_R3. Usage of heat from the building's waste water for HP which heats up the building's recirculation pipes network of domestic hot water.

The main technical indicators of a heat user are used in these calculations:

There are ten apartments in a stairwell (total area ~ 475 m²) from which the ventilation system (natural or mechanical) extracts 1600 m³/h of air. The temperature of the exhaust air is 18 – 20°C. There are 30 residents who use 60 m³ of cold water and 22 m³ of domestic hot water a month. The heat flow of the non-insulated stairwell's domestic hot water pipes and towel dryers and insulated stairwell's main pipe network is equal to 2.7 kW. The heat flow of the non-insulated stairwell's towel dryers and insulated domestic hot water pipes, as well as the insulated main stairwell's pipe network, is equal to 1.5 kW. If domestic hot water system in a stairwell is renovated and water consumption level remains the same, it needs 25 MWh of heat per year, 13 of which are used to heat the water.

A. Alternative: HP recovering heat from exhaust air, HP_R1

Recovering heat from mechanical exhaust air when there is no mechanical air supply system and HP is installed in bivalent (operating together with DHS) in semi-accumulative domestic hot water preparation system in a stairwell. For the preparation of domestic hot water and maintaining the temperature in the insulated recirculation pipe network in a stairwell, a semi-accumulative domestic hot water preparation system is chosen with a water heater of the capacity 875 litres, where a 12 kW water heating coil is installed. Non-accumulative hot water system using HP is not analysed.

The technological scheme of HP_R1 and technical indicators are shown in Figure 1.

The technical parameters of HP_R1a and HP_R1b alternatives using refrigerant R152a are presented in Table II.

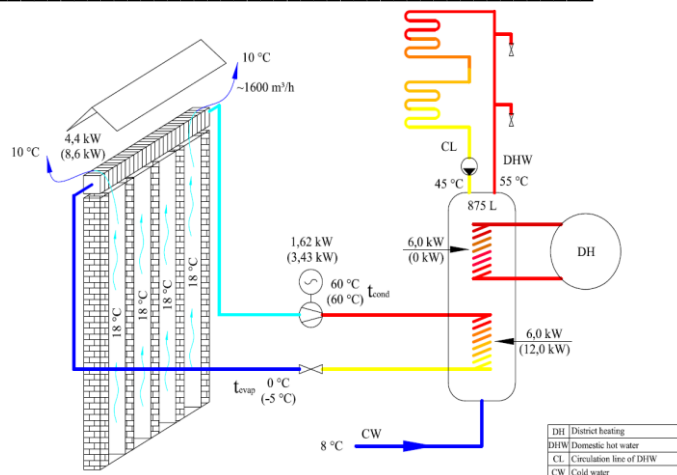


Fig. 1. Technological circuit of domestic hot water system by using HP recovering heat from exhaust air.

TABLE II
PARAMETERS OF HP SYSTEM FOR HP_R1A (50%) AND HP_R1B (100%) ALTERNATIVES

	Compressor	COP	Evaporator			Condenser		
			kW	-	kW	$t_{\text{evap}}, ^\circ\text{C}$	$\Delta t_{\text{min}}, ^\circ\text{C}$	kW
HP_R1a	1,62	3,7	4,4	0	10	6,0	60	5
HP_R1b	3,43	3,5	8,6	-5	7,5	12,0	60	5

In the first (HP_R1a) case, a half of the capacity required by DHW system is produced by HP. The operation span of HP and DHS may vary on a daily and on a yearly basis.

In the second (HP_R1b) case, DHS heat is not used for DHW. Besides, in this case, DHS still supplies heat to the building's heating system. Δt_{min} indicate the minimal temperature difference between exhaust air and refrigerant in an evaporator and the minimum temperature difference among heated water and refrigerant in a condenser. The technically minimal values are chosen because, otherwise, HP indicators would decline or the mentioned heat exchangers would be unacceptably large.

B. Alternative: HP in the air handling unit, HP_R2

Heat from mechanical exhaust air utilisation by installing HP in an air handling unit instead of a preheating section after or instead of plate recuperator in a stairwell.

In the HP_R2 alternative, HP is installed in an air handling unit instead of a preheating section after the plate heat exchanger (exhaust air heat recovery exchanger – EAHRE).

In this case, this solution was analysed for one stairwell of a multi-apartment building. The technological calculations have been made at two external air temperatures, i.e. when the external air temperatures are -23 °C (low temperature) and 0 °C (mild temperature). The results show that the operation mode of HP should be

related to the external air temperature. The mild temperature is close to the temperature of the heating season which lasts for slightly more than 200 days. The calculations cover 210 days, 110 of which have the temperature of 0 °C and more. The average value of the low temperature period is -5 °C, and of the mild temperature period is 5.4 °C. Technological issues of such regulation are not analysed in this article. This might be solved by using variable speed or two-stage compressors. Without such regulation, additional capacity of the produced heat is generated, but its usage possibilities require a more detailed analysis of the building's heat flows. Before making a decision to install HP, it is advisable to make an analysis of its produced heat capacity during the entire heating season, by considering the local external air temperature. There is a lack of valid information on ventilation units' heat exchanger seasonal temperature transfer efficiency, which is certainly lower than the nominal or declared efficiency by several tens of a per cent. The research performed by the Department of Building Energetics of the Vilnius Gediminas Technical University confirms these assumptions [20]. Table III presents the temperatures of the heat exchanger and the temperature transfer efficiency for the alternative HP_R2.

TABLE III
TEMPERATURE OF RECOVERED HEAT AND TEMPERATURE TRANSFER EFFICIENCY OF ALTERNATIVE HP_R2

		HP_R2a ($\eta_t=0,60/0,50$)		HP_R2b ($\eta_t=0,75/0,65$)	
		-23 to 0°C	$\geq 0^\circ\text{C}$	-23 to 0°C	$\geq 0^\circ\text{C}$
Cold supplied air	t_1	-23.0	0.0	-23.0	0.0
Hot supplied air	t_2	0.6	9.2	7.2	12.1
Hot exhaust air	t_3	19.0	19.0	19.0	19.0
Cold exhaust air	t_4	-6.2	9.5	-12.5	6.6
Temperature transfer efficiency	η_t	0.60	0.50	0.75	0.65

When a heater of the air handling unit is changed by HP, the air of t_2 temperature from the heat recovery unit goes to the HP condenser. Afterwards, the cooled exhaust air t_4 goes to the HP evaporator.

In the alternative HP_R2c, there is no heat recovery unit, instead the exhaust air with temperature t_3 goes directly to the HP evaporator. The air exhaust from the evaporator shall have the temperature $t_{4\text{evap}}$. Then t_1 temperature external air is supplied directly to the condenser. The air exhaust from the condenser and supplied to the premises shall always have temperature $t_{2\text{cond}}$.

The technological scheme of alternative HP_R2 is shown in Figure 2.

The technological calculation results for one stairwell (1600 m³/h of air) of the combinations of exhaust air heat

recovery HP (EAHRHP) with heat recovery unit of the air handling unit (alternative HP_R2) are presented in Table IV.

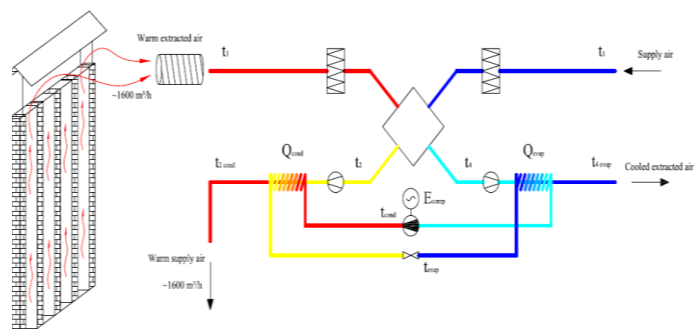


Fig. 2. Exhaust Air Heat Recovery Heat Pump-EAHRHP in air handling unit.

TABLE IV
EXHAUST AIR HEAT RECOVERY HP IN AIR HANDLING UNIT - AN ALTERNATIVE HP_R2.

	Compressor	COP	Evaporator			Condenser		
	kW	-	kW	$t_{\text{evap}}, ^\circ\text{C}$	$\Delta t_{\text{min}}, ^\circ\text{C}$	kW	$t_{\text{cond}}, ^\circ\text{C}$	$\Delta t_{\text{min}}, ^\circ\text{C}$
HP_R2a ($t_1=-23$ to 0 °C)	3.06	3.53	7.76	-30.0	11.0	10.8	30.0	10.0
HP_R2am ($t_1 \geq 0$ °C)	1.08	5.47	4.83	-10.0	11.0	5.91	30.0	10.0
HP_R2b ($t_1=-23$ to 0 °C)	2.07	3.54	5.25	-30.0	9.3	7.32	30.0	10.0
HP_R2bm ($t_1 \geq 0$ °C)	0.79	5.47	3.53	-10.0	10.5	4.32	30.0	10.0
HP_R2c ($t_1=-23$ to 0 °C)	7.14	3.53	18.1	-30.0	16.0	25.2	30.0	10.0
HP_R2cm ($t_1 \geq 0$ °C)	2.10	5.48	9.4	-10.0	12.0	11.5	30.0	10.0

Technologically, in the air handling unit HP which partially or fully changes the heat recovery unit and heater may operate with COP approx. 3.5 under low temperatures ($t_1=-23$ ° up to 0 °C) and with COP approx. 5.5 under mild temperatures ($t_1 \geq 0$ °C), irrespective of the temperature transfer efficiency of heat recovery unit. Thus, under higher temperature, COP increases about 1.5 times and the capacity of the compensator decreases by 2.8 times. Naturally, the increasing temperature transfer efficiency of the heat recovery unit requires HP with a lower capacity. When the temperature transfer efficiency increases from 0.6 up to 0.7 (i.e. by 1.25 times), HP is required, i.e. the capacity of its compressor declines by approx. 1.45 times.

C. Alternative: Waste water resource heat pump, HP_R3

In alternative HP_R3. heat from waste water of the building with 4 stairwells and 40 apartments is used for HP which heats the domestic hot water by recirculation. In this case, all the waste water from the building is collected in one receptacle where the HP evaporator is installed. A separate receptacle for each stairwell would require additional investments and work. An example of such a receptacle with evaporator is [21].

The technological scheme and technical parameters of HP_R3 or the so-called waste water resource HP (WWRHP) alternative are shown in Figure 3.

It is assumed that the existing DHW recirculation pipes are partially insulated (except of towel dryers) and because of that, the heat flow is reduced by two times.

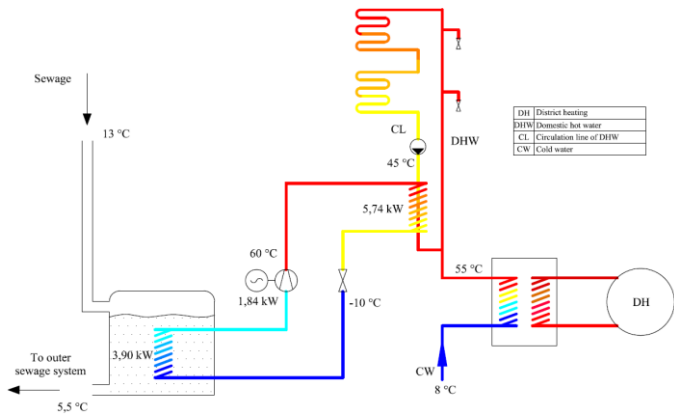


Fig. 3. Waste water resource heat pump (WWRHP) – principle technological scheme and technical indicators of the alternative HP_R3.

IV. ENERGY, ECONOMIC, AND ENVIRONMENTAL EVALUATION OF ALTERNATIVES

The so-called 3E evaluation has been performed. The aggregate energy, economic and environmental indicators of the alternatives are presented in Table V. The comments and not yet discussed basic assumptions are presented below.

Heat price from DHS is 22.4 ct/kWh, incl. VAT, and electricity price - 45 ct/kWh, incl. VAT.

HP_R1a system for a stairwell (bivalent water heating when HP is 6 kW and DHS – 6 kW, twin coil accumulation tank of 875 litres) costs LTL 31.5 thousand. (66 LTL/m²). HP_R1b system for a stairwell (water heated only by HP when HP is 12 kW, twin coil accumulation tank of 875 litres) costs LTL 34.3 thousand (72.2 LTL/m²).

TABLE V
AGGREGATE INDICATORS OF ALTERNATIVES

Alternative	COP	Produced heat		Investments to heated area	Cost of heat produced by HP		Simple payback time	CO ₂ decrease
		HP	DH* or EAHR**		ct/kWh	% of DHS price		
		MWh		LTL***/m ²			year	%
HP_R1a1	3.7	16.7	8.3*	66.2	12.15	54.2	19	14.8
HP_R1a2	3.7	12.5	12.5*	66.2	12.15	54.2	25	11.1
HP_R1b	3.5	25.0	0.0*	72.2	12.86	57.3	15	17.7
HP_R2a	3.53	14.5	19.2**	57.0	12.75	56.8	9	16.9
HP_R2am	5.47	11.3	10.0**		8.23	36.7		
HP_R2b	3.53	9.8	23.9**	50.0	12.75	56.8	11	12.3
HP_R2bm	5.47	8.4	12.9**		8.23	36.7		
HP_R2c	3.53	33.7	0.0**	63.0	12.75	56.8	5	33.5
HP_R2cm	5.47	21.3	0.0**		8.23	36.7		
HP_R3	3.10	49.4	0.0*	14.2	14.52	64.7	7	7.0

***1€ = 3.4528 LTL

Alternative HP_R2 – the use of mechanically exhaust air heat from each stairwell when there is mechanical air supply, installing HP in an air handling unit instead of preheating section after (a and b) and instead of (c) plate recuperator. Its technological scheme is shown in Figure 2. As Table V shows, the payback time of alternative HP_R2a is 9 years. It should be noted that in alternative HP_R2c/2cm the compared ventilation system is without heat recovery. Thus, when the reference variant is highly inefficient, the results of the alternative variant at first approach look quite satisfactory.

In alternative HP_R3, HP is installed for the whole building and it uses a R152a cooler. The characteristics of this HP: the evaporator of 3.90 kW, the compressor of 1.84 kW, the condenser of 5.74 kW, and its COP = 3.10. The temperature of sewage flowing into the collector with evaporator is 13°C and the one flowing out of the collector – 5.5°C. This system for the building would cost approx. LTL 27 thousand, incl. VAT (14.2 LTL/m²). The operation of a well insulated recirculation domestic hot water loop of DHW

requires 50 MWh of heat per year. The installation of this system would save LTL 3.9 thousand per year. The payback period reaches almost 7 years.

This system could operate together with systems HP_1 or HP_2, besides, it operates all year round.

The environmental evaluation could be based on the absolute and percentage (counted for the production of heat capacity) decrease of CO₂. There is an assumption that when heat is produced using natural gas (80%), fuel oil (10%), and biomass (10%), the heat emission factor, depending on the fuel, is 0.22 kg_{CO2}/kWh. For the evaluation of the pollution of the used electricity, the Lithuanian power-station emission factor 0.634 kg_{CO2}/kWh is used. The analysed alternatives of HP installation in five building services systems allow reducing CO₂ by approx. 10-20%.

It should be mentioned that, from the environmental point of view, none of the analysed alternatives of HP installation in building services systems are better than DHS where biomass covers approx. 50% and more of fuel balance.

V. CONCLUSIONS

1. As the performed analysis shows, if the thermal characteristics of new or renovated buildings increase, air as well as water warm flows from the building, change very slightly, thus they should receive more attention for energy efficiency of their usage and particularly for HP applications.

2. Installation of HP in the air handling unit, where it uses heat from exhaust air, and in the waste water receptacle, where it uses heat of this water, from the payback period point of view is economically attractive and is worthwhile to be supported in the pilot projects. In each case, the size and the operation strategy of HP must be calculated under specific climatic and operating conditions.

3. According to the simplified environmental assessment based on a CO₂, HP does not have an advantage over DHS which burns great quantities of biomass. In order to evaluate all the environmental effects, it is necessary to use more sophisticated methods of assessment.

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Tehniskie risinājumi siltuma sūkņu uzstādīšanai privātmājās ir labi pazīstami, bet to integrācija jau esošās sistēmās daudzdzīvokļu ēkās notiek diezgan reti. Siltuma sūkņa tehnoloģiju bieži sauc par atjaunojamo, bet to integrācija daudzdzīvokļu ēkās esošajā siltumapgādes sistēmā sekmē tehnisko, ekonomisko un ekoloģisko izaugsmi kopumā. Darba mērķis ir siltuma sūkņu integrācijas iespēju izpēte esošajās siltuma/sanitārajās sistēmās daudzstāvu dzīvojamās ēkās. Vismazākā telpa, kurā iespējams integrēt siltuma sūkni, ir daudzstāvu ēkas kāpņu telpa. Siltuma sūkņu izmantošanas iespējas atsevišķā dzīvoklī netika aplūkotas. Detalizēti tika analizētas trīs alternatīvas: HP_R1 – siltuma sūkņī saražotā gaisa siltuma izmantošana karstā ūdens sagatavošanai; HP_R2 – siltuma sūkņī saražotā gaisa siltuma izmantošana telpu apsildei, pieslēdzoties rekuperatora vai mehāniskās ventilācijas sistēmai; HP_R3 – ēkas notekūdeņu siltuma izmantošana siltuma sūkņī karstā ūdens sagatavošanai. Alternatīvu analīze un aprēķini tika veikti reālām daudzdzīvokļu ēkām Brištonas pilsētā (Lietuva) projekta CONCERTO Eco-Life ietvaros. Analīzē iekļauti arī alternatīvu ekonomiskie un ekoloģiskie novērtējumi.